

Ball Bearings Shield and Seal Types

CAT.No.3015-5/E



Ball Bearings

Shield and Seal Types

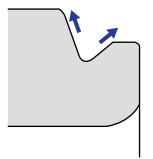


Modern machinery is becoming smaller and more compact. As a result, the use of integral seals or shields on ball bearings enables machinery designers to achieve their downsizing goals while also extending maintenance intervals.

1

Dynamic seal groove results in excellent performance (all types)

A V-shaped groove is cut around the circumference of the inner ring and acts as the dynamic surface for the seal or shield. Due to centrifugal force, the outer surface of the v acts to keep out dirt; the inner surface of the v acts to seal in the grease.



2

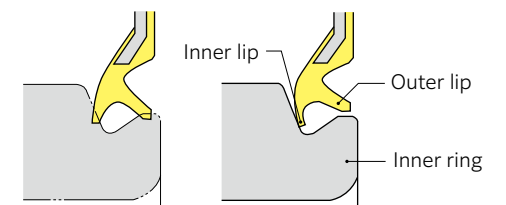
Rust resistant shield (ZZ spec)

Shields are simple metallic devices and result in low torque characteristics. Shields have a corrosion resistant coating to ensure long life when exposed to the elements.

3

Double-lip seal adjusts for wear (LLU spec)

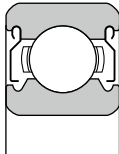
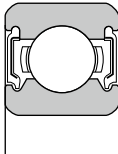
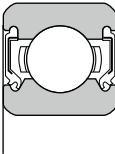
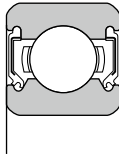
The inner lip of a double-lip seal is pressed against the inside surface of the V-groove; the outer lip is spread open by the elastic force of rubber to create a small gap between the outer lip and the inner ring. Should the inner lip wear, the contact pressure will decrease; however, the outer lip will get closer to the inner ring. In effect, a constant degree of sealing will be maintained.





Non-contact shield type (ZZ) Non-contact seal type (LLB) Contact seal type (LLU) Low torque contact type (LLH)

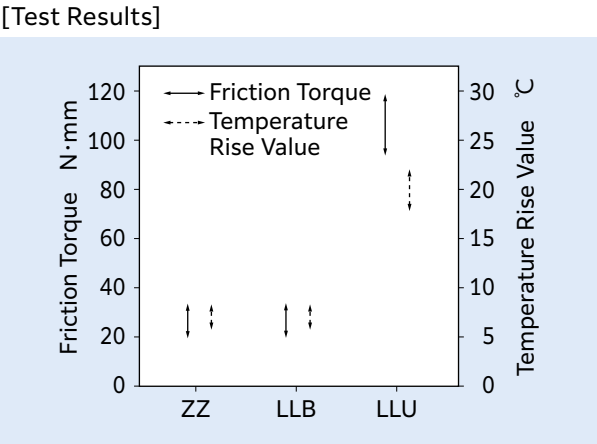
Table 1 Bearing design and features

Types and codes		Non-contact shield type	Non-contact seal type	Contact seal type	Low torque contact type
		ZZ	LLB	LLU	LLH
Construction					
	• Metal shield plate is affixed to the outer ring; the inner ring incorporates a V-groove and labyrinth clearance.	• The outer ring incorporates synthetic rubber molded to a steel plate; seal edge is aligned with V-groove along inner ring surface with labyrinth clearance.	• The outer ring incorporates synthetic rubber molded to a steel plate; seal edge contacts V-groove along inner ring surface.	• Basic construction is the same as LLU-type, but specially designed lip on the edge of the seal prevents foreign matter penetration; low torque construction.	
Performance comparison	Torque	Low	Low	Higher	Medium
	Dust proofing	Good	Better than ZZ-type	Excellent	Much better than LLB-type
	Water proofing	Poor	Poor	Very good	Good
	High speed capacity	Same as open type	Same as open type	Limited by contact seals	Much better than LLU-type
	Allowable temp. range ¹⁾	Depends on lubricant	-20 to 120 °C	-20 to 110 °C	-20 to 120 °C

1) Please consult NTN Engineering about applications which exceed the allowable temperature range of products listed on this table.
Note: This chart lists double shielded and double sealed bearings, but single shielded (Z) and single sealed (LB, LU, LH) are also available. Grease lubrication should be used with single shielded and single sealed bearings.

Friction Torque and Temperature Rise Test

- [Test Conditions]
- Bearing: 6305
 - Grease: Alvania grease S3 (3.5 g filling)
 - Rotational speed: 1 800 min⁻¹
 - Time: 20 minutes
 - Load: Radial load 29 N
 - Quantity tested: 5 pcs. each



Note: The temperature rise value will vary depending on the heat dissipation conditions. In actual use, the temperature will be lower than the above value due to heat dissipation from the housing.

Fig. 1 Friction torque and temperature rise

Dust Test

- [Test Conditions]
- Bearing: 6305
 - Grease: Alvania grease S3 (3.5 g filling)
 - Rotational speed: 1 800 min⁻¹
 - Load: Radial load 29 N
 - Volume of testing machine: 40 liters
 - Dust: Oxidized ferrous powder (2 kg) + (1 kg) mixture
 - Test time: 1 hour

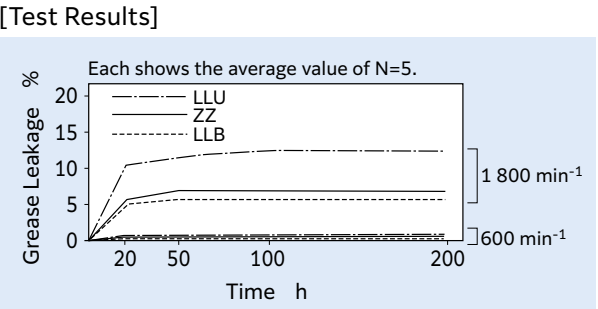
Table 2 Test results

Seal type	No.	1	2	3	4	Average
ZZ		2.5	2	2	2.5	2.25
LLB		2.5	3	2.5	2.5	2.62
LLU		3	3	3	3	3.00

Ratings 3: Complete absence of dust
 2: Slight presence of dust observed
 1: Presence of dust clearly observed

Grease Leakage Test

- [Test Conditions]
- Bearing: 6305
 - Grease: Alvania grease S3 (3.5 g filling)
 - Rotational speed: 1 800 min⁻¹, 600 min⁻¹
 - Ambient temperature: 80 °C
 - Time: 200 hours
 - Quantity tested: 5 pcs. each



Note: The larger grease leakage with the LLU at 1 800 min⁻¹ is due to bearing temperature rise.

Fig. 2 Grease leakage

Muddy Water Splash Test

- [Test Conditions]
- Bearing: 6304
 - Grease: Alvania grease S3 (2 g filling)
 - Rotational speed: 3 000 min⁻¹
 - Muddy water: 5 wt% of class 8 of JIS Z 8901 Kanto loam powder
 - Muddy water flow: 30ml/min
 - Test time: 10 cycles 240 hours
(1 cycle: 3 hours of spraying and 21 hours of drying)

Table 3 Test results

Seal type	No.	1	2	3	4	5	Average
LLB		2	1	1	2	2	1.6
LLU		3	2	2	4	3	2.8

Ratings 4: No penetration whatsoever
 3: Very slight penetration of muddy water is apparent
 2: Some penetration of muddy water is apparent
 1: Considerable penetration of muddy water is apparent

Operating Life of Pre-lubricating Grease

The operating life of filled grease is influenced by many factors: operating conditions, temperature, atmosphere, etc. There is even a considerable difference by grease brands. Keeping this in mind, Fig.3 presents a scale for determining grease life. It is based on actual and experimental data; and applies to high-quality grease used in ball bearings under normal operating conditions. As the operating temperature increases, the grease life, of course, becomes shorter. Although not accurate for all greases, Fig.3 is valid, in general guidelines, for operating temperatures up to 80 °C; for operating temperatures over 80 °C. Operating life decreases to about two-thirds for every 10 °C over 80 °C.

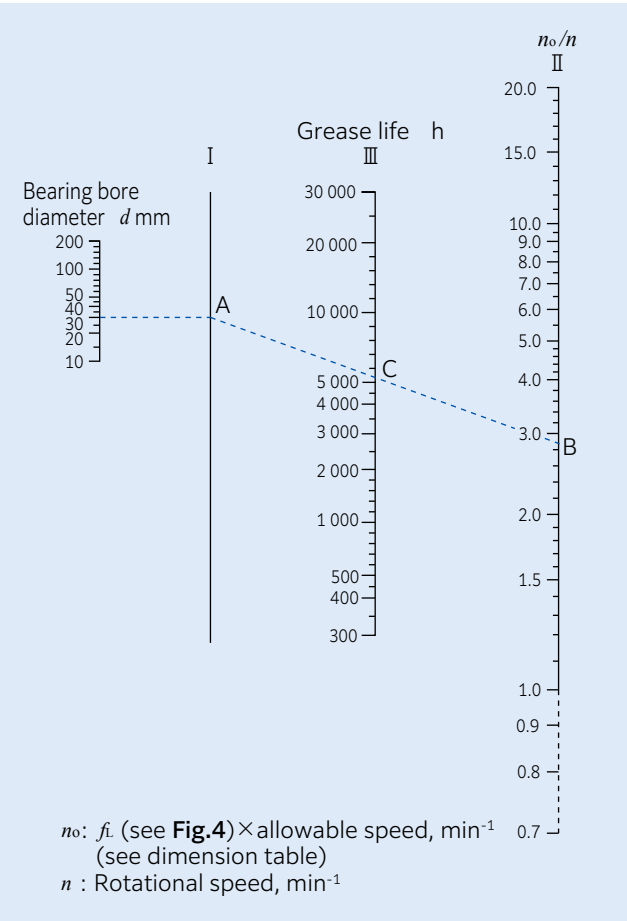


Fig. 3 Diagram for grease life

[Example] Find the grease life for deep groove ball bearing 6206, with a radial load of 2.0 kN, rotational speed at 3 600 min⁻¹

The Allowable rotational speed from dimensions tables for bearing 6206 is 11 000 min⁻¹. From Fig.4 $C_r/P_r = 21.6/2.0 \text{ kN} = 10.8$, $f_L = 0.96$. Allowable rotational speed n_o for 2.0 kN radial load is:

$n_o = 0.96 \times 11\,000 = 10\,560 \text{ min}^{-1}$

Therefore, $\frac{n_o}{n} = \frac{10\,560}{3\,600} = 2.93$

The point where vertical line I intersects a horizontal line drawn from the point equivalent of $d = 30$ for the bearing bore shown in Fig.3 shall be point A. Find intersection point C where vertical line III intersects the straight line formed by joining point B ($n_o/n = 2.93$) with A by a straight line II. It shows that grease life in this case is approximately 5 500 hours.

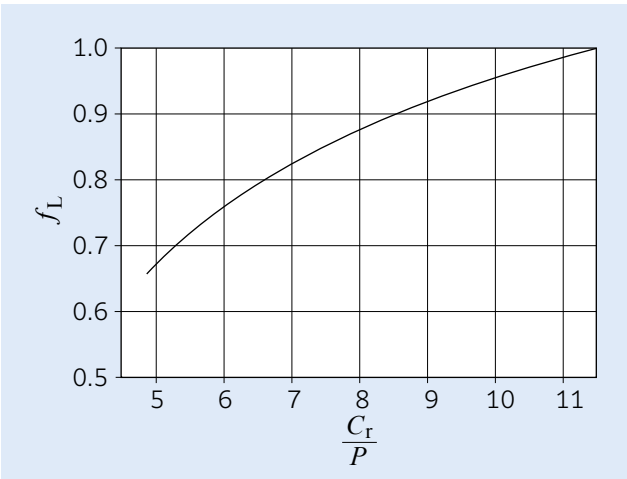


Fig. 4 Value of adjustment factor f_L depends on bearing load

Note that the performance of ball bearings shield and seal types is affected by operating conditions and grease type. The following conditions, please contact NTN Engineering.

- Use under high or low temperature
- Low torque is required
- High speeds are required
- Grease leakage must be avoided
- Use under other special conditions

Fatigue Load Limit (C_u)

The fatigue load limit is the applied load on a bearing that results in just reaching the fatigue stress limit at the maximum loaded raceway contact. This depends on the bearing type, internal specifications, quality, and material strength. In ISO 281:2007, 1.5 GPa is recommended as the fatigue stress limit corresponding to C_u for bearings made of commonly used high quality material and good manufacturing quality. Values for the fatigue load limit with respect to the NTN bearing numbers are provided in the dimensional table. The life modification factor, a_{iso} , should be evaluated considering the fatigue load limit. For details see catalog “Ball and Roller Bearings (CAT. No.2203/E) section 3.4 Modified rating life”.

Bearing Fits

General standards of bearing fits for each condition of use are shown in Table 4 through Table 6.

Table 4 Tolerance class of shafts commonly used for deep groove ball bearings (Classes 0 and 6)

Conditions		Shaft diameter (mm)		Shaft tolerance class	Remarks
		Over	Incl.		
Rotating Inner ring load or Load of undermined direction	Light load ¹⁾ or Fluctuating load	—	18	h5	When greater accuracy is required js6 and k6, may be substituted for js5 and k5.
		18	100	js6	
		100	200	k6	
	Normal load ¹⁾	—	18	js5	—
Inner ring static load	Inner ring must move easily over shaft	18	100	k5	
		100	140	m5	
		140	200	m6	
	Inner does not have to move easily over shaft	Overall shaft diameter		h6	
Center axial load		Overall shaft diameter		js6	Generally, shaft and inner rings are not fixed using resultant fits.

1) Standards for light loads and normal loads
Light loads : dynamic equivalent radial load $\leq 0.05C_r$
Normal loads : $0.05C_r < \text{dynamic equivalent radial load} \leq 0.10C_r$
Note: All values and fits listed in the above tables are for solid steel shafts.

Table 5 Tolerance class of housing bore commonly used for deep groove ball bearings (Classes 0 and 6)

Conditions			Housing bore tolerance class	Remarks	
Housing	Load types, etc.	Outer ring axial direction movement ³⁾			
Single housing or Split housing	Static outer ring load	All types of loads	Yes	H7	G7 can be used for large bearings or bearings with large temperature differential between the outer ring and housing.
		Light load ¹⁾ or normal load ¹⁾	Yes	H8	—
		Shaft and inner ring become hot	Easily	G7	F7 can be used for large bearings or bearings with large temperature differential between the outer ring and housing.
Single housing		Requires precise rotation under light or normal loads	As a rule, cannot move.	K6	—
			Yes	JS6	—
		Requires low noise operation	Yes	H6	—
	Indeterminate load	Light load or normal load	Yes	JS7	If high accuracy is required, JS6 and K6 are used in place of JS7 and K7.
		Normal load or heavy load ¹⁾	As a rule, cannot move.	K7	
		High impact load	No	M7	
	Rotating outer ring load	Light load or fluctuating load	No	M7	—
		Normal load or heavy load	No	N7	Primarily applies to ball bearings.
		Heavy load or large impact load with thin wall housing ²⁾	No	P7	—

1) Standards for light loads, normal loads, and heavy loads
Light loads : dynamic equivalent radial load $\leq 0.05C_r$
Normal loads : $0.05C_r < \text{dynamic equivalent radial load} \leq 0.10C_r$
Heavy loads : $0.10C_r < \text{dynamic equivalent radial load}$
2) The axial direction needs to be secured because the outer ring may move in the shaft direction, causing problems, depending on the use (Example: planetary gear, etc.).
3) Indicates whether or not outer ring axial movement is possible with non-separable type bearings.
Note: 1 All values and fits listed in the above tables are for cast iron or steel housings.
2 If only a center axial load is applied to the bearing, select a tolerance class that provides clearance for the outer ring in the radial direction.

Table 6 Fits for electric motor bearings

Bearing type	Shaft fits		Housing fits	
	Shaft diameter (mm)	Tolerance class	Housing bore diameter	Tolerance class
Deep groove ball bearings	Over	Incl.	All sizes	H6 or J6
	—	18		
	18	100		
	100	160		

Table 7 Inner rings

Nominal bearing bore diameter		Deviation of mean bore diameter in a single plane								Variation of bore diameter in a single plane															
d		Δ_{dmp}								V_{dsp}															
mm										Bearing series 68, 69								Bearing series 60							
		Class 0		Class 6		Class 5		Class 4 ¹⁾		Class 0	Class 6	Class 5	Class 4	Class 0	Class 6	Class 5	Class 4	Class 0	Class 6	Class 5	Class 4	Class 0	Class 6	Class 5	Class 4
Over Incl.		Upper	Lower	Upper	Lower	Upper	Lower	Upper	Lower	Max.				Max.				Max.				Max.			
2.5	10	0	-8	0	-7	0	-5	0	-4	10	9	5	4	8	7	4	3	6	5	4	3	6	5	4	3
10	18	0	-8	0	-7	0	-5	0	-4	10	9	5	4	8	7	4	3	6	5	4	3	6	5	4	3
18	30	0	-10	0	-8	0	-6	0	-5	13	10	6	5	10	8	5	4	8	6	5	4	8	6	5	4
30	50	0	-12	0	-10	0	-8	0	-6	15	13	8	6	12	10	6	5	9	8	6	5	9	8	6	5
50	80	0	-15	0	-12	0	-9	0	-7	19	15	9	7	19	15	7	5	11	9	7	5	11	9	7	5
80	120	0	-20	0	-15	0	-10	0	-8	25	19	10	8	25	19	8	6	15	11	8	6	15	11	8	6
120	150	0	-25	0	-18	0	-13	0	-10	31	23	13	10	31	23	10	8	19	14	10	8	19	14	10	8
150	180	0	-25	0	-18	0	-13	0	-10	31	23	13	10	31	23	10	8	19	14	10	8	19	14	10	8

1) The dimensional difference Δd s of the measured bore diameter applied to Class 4 is the same as the tolerance of dimensional difference Δd_{mp} of the mean bore diameter within a plane. However, the dimensional difference is applied to bearing series 60, 62 and 63 for Class 4.

Table 8 Outer rings

Nominal bearing outside diameter		Deviation of mean outside diameter in a single plane								Variation of outside diameter in a single plane ³⁾				Variation of mean outside diameter				Radial runout of outer ring of assembled bearing			
D		Δ_{Dmp}								V_{Dsp}				V_{Dmp}				K_{ea}			
mm		Class 0		Class 6		Class 5		Class 4 ²⁾		Bearing series 62, 63		60,62,63		Class 0		Class 6		Class 0		Class 6	
Over Incl.		Upper	Lower	Upper	Lower	Upper	Lower	Upper	Lower	Max.		Max.		Max.		Max.		Max.		Max.	
18	30	0	-9	0	-8	0	-6	0	-5	10	9	7	6	3	2.5	15	9	6	4	15	9
30	50	0	-11	0	-9	0	-7	0	-6	12	10	8	7	4	3	20	10	7	5	20	10
50	80	0	-13	0	-11	0	-9	0	-7	16	13	10	8	5	3.5	25	13	8	5	25	13
80	120	0	-15	0	-13	0	-10	0	-8	20	16	11	10	5	4	35	18	10	6	35	18
120	150	0	-18	0	-15	0	-11	0	-9	26	20	14	11	6	5	40	20	11	7	40	20
150	180	0	-25	0	-18	0	-13	0	-10	30	25	19	14	7	5	45	23	13	8	45	23
180	250	0	-30	0	-20	0	-15	0	-11	38	30	23	15	8	6	50	25	15	10	50	25
250	315	0	-35	0	-25	0	-18	0	-13	—	—	26	19	9	7	60	30	18	11	60	30

2) The dimensional difference ΔD s of the measured outside diameter applied to Class 4 is the same as the tolerance of dimensional difference ΔD_{mp} of the mean outside diameter within a plane. However, the dimensional difference is applied to bearing series 60, 62 and 63 for Class 4.

3) Applies to cases where snap rings are not installed on the bearings.

Radial Internal Clearance

Table 9 Radial internal clearance

Nominal bearing bore diameter		C2		CN		C3		C4		C5	
d mm		Min.	Max.	Min.	Max.	Min.	Max.	Min.	Max.	Min.	Max.
Over	Incl.										
6	10	0	7	2	13	8	23	14	29	20	37
10	18	0	9	3	18	11	25	18	33	25	45
18	24	0	10	5	20	13	28	20	36	28	48
24	30	1	11	5	20	13	28	23	41	30	53
30	40	1	11	6	20	15	33	28	46	40	64
40	50	1	11	6	23	18	36	30	51	45	73
50	65	1	15	8	28	23	43	38	61	55	90
65	80	1	15	10	30	25	51	46	71	65	105
80	100	1	18	12	36	30	58	53	84	75	120
100	120	2	20	15	41	36	66	61	97	90	140
120	140	2	23	18	48	41	81	71	114	105	160
140	160	2	23	18	53	46	91	81	130	120	180
160	180	2	25	20	61	53	102	91	147	135	200

Unit: μm

Variation of mean bore diameter				Radial runout of inner ring of assembled bearing				Perpendicularity of inner ring face with respect to the bore		Axial runout of inner ring of assembled bearing		Deviation of a single inner ring width				Variation of inner ring width			
V_{dmp}				K_{ia}				S_d		S_{ia}		Δ_{Bs}				V_{Bs}			
Class 0	Class 6	Class 5	Class 4	Class 0	Class 6	Class 5	Class 4	Class 5	Class 4	Class 5	Class 4	Class 0	Class 5	Class 0	Class 5	Class 0	Class 6	Class 5	Class 4
Max.				Max.				Max.		Max.		Upper	Lower	Upper	Lower	Max.			
6	5	3	2	10	6	4	2.5	7	3	7	3	0	-120	0	-40	15	15	5	2.5
6	5	3	2	10	7	4	2.5	7	3	7	3	0	-120	0	-80	20	20	5	2.5
8	6	3	2.5	13	8	4	3	8	4	8	4	0	-120	0	-120	20	20	5	2.5
9	8	4	3	15	10	5	4	8	4	8	4	0	-120	0	-120	20	20	5	3
11	9	5	3.5	20	10	5	4	8	5	8	5	0	-150	0	-150	25	25	6	4
15	11	5	4	25	13	6	5	9	5	9	5	0	-200	0	-200	25	25	7	4
19	14	7	5	30	18	8	6	10	6	10	7	0	-250	0	-250	30	30	8	5
19	14	7	5	30	18	8	6	10	6	10	7	0	-250	0	-250	30	30	8	5

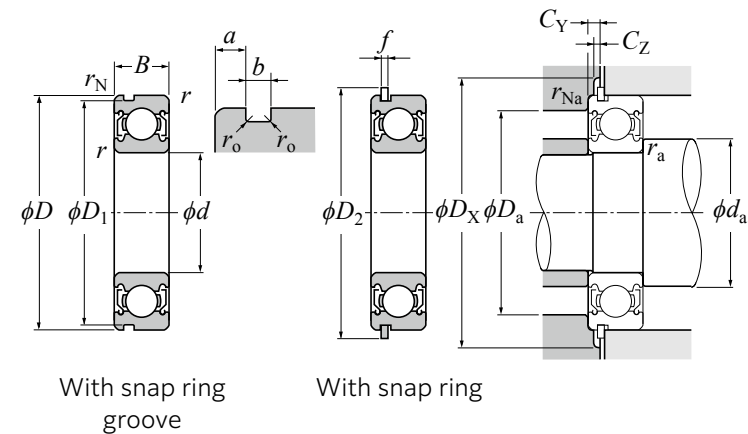
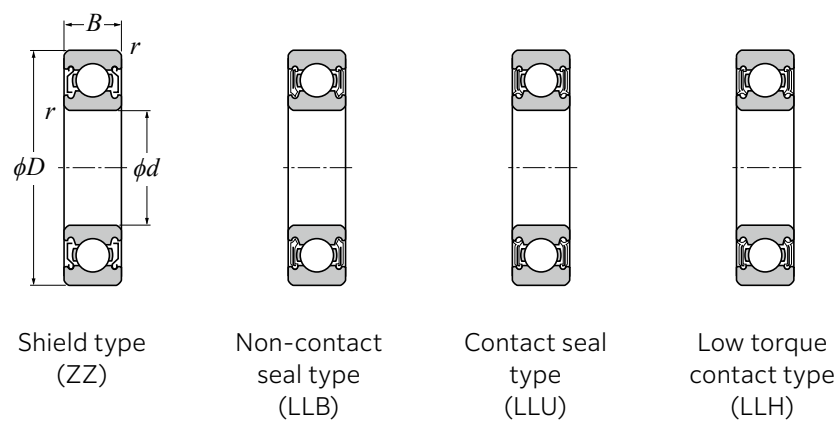
Unit: μm

Perpendicularity of outer ring outside surface with respect to the face		Axial runout of outer ring of assembled bearing		Deviation of a single outer ring width	Variation of outer ring width			
S_D		S_{ea}		ΔC_s	V_{C_s}			
Class 5	Class 4	Class 5	Class 4	All classes	Class 0	Class 6	Class 5	Class 4
Max.		Max.			Max.			
8	4	8	5	Depends on tolerance of ΔB_s in relation to d of the same bearing			5	2.5
8	4	8	5				5	2.5
8	4	10	5		Depends on tolerance of V_{B_s} in relation to d of the same bearing		6	3
9	5	11	6				8	4
10	5	13	7				8	5
10	5	14	8				8	5
11	7	15	10				10	7
13	8	18	10				11	7

Table 10 Radial internal clearance of bearings for electric motor

Nominal bearing bore diameter		CM	
d mm		Min.	Max.
Over	Incl.		
10 _(incl.)	18	4	11
18	24	5	12
24	30	5	12
30	40	9	17
40	50	9	17
50	65	12	22
65	80	12	22
80	100	18	30
100	120	18	30
120	140	24	38
140	160	24	38
160	180	—	—

Dimension Table



Dynamic equivalent radial load
 $P_r = X F_r + Y F_a$

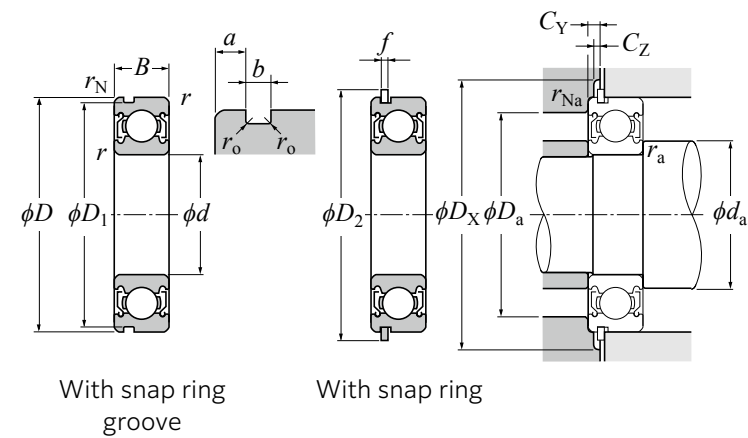
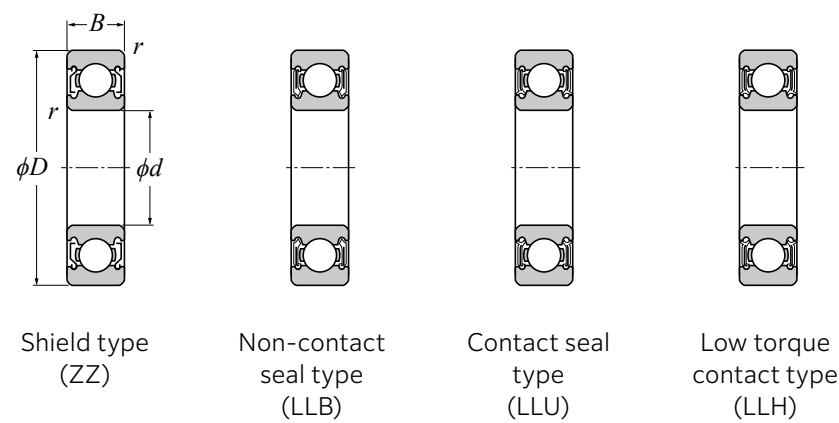
$\frac{f_0 \cdot F_a}{C_{0r}}$	e	$\frac{F_a}{F_r} \leq e$		$\frac{F_a}{F_r} > e$	
		X	Y	X	Y
0.172	0.19				2.30
0.345	0.22				1.99
0.689	0.26				1.71
1.03	0.28				1.55
1.38	0.30	1	0	0.56	1.45
2.07	0.34				1.31
3.45	0.38				1.15
5.17	0.42				1.04
6.89	0.44				1.00

Static equivalent radial load
 $P_{0r} = 0.6 F_r + 0.5 F_a$
When,
 $P_{0r} < F_r$ use $P_{0r} = F_r$.

d 10-20 mm

Boundary dimensions			Basic load rating		Fatigue load limit	Factor	Allowable speed				Bearing number			
mm			kN		kN		min ⁻¹							

Dimension Table



Dynamic equivalent radial load
 $P_r = X F_r + Y F_a$

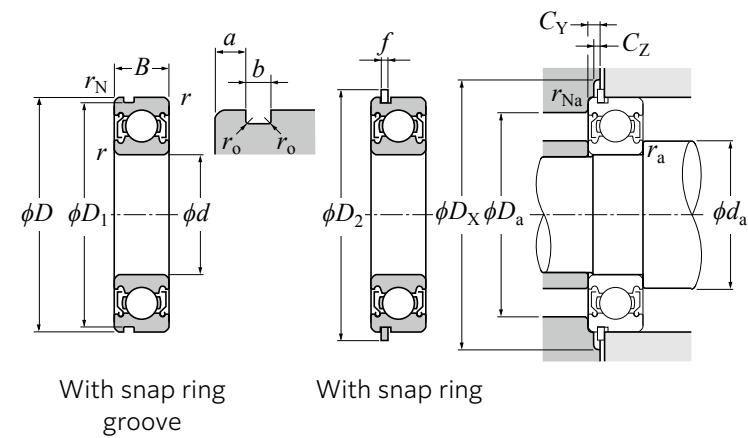
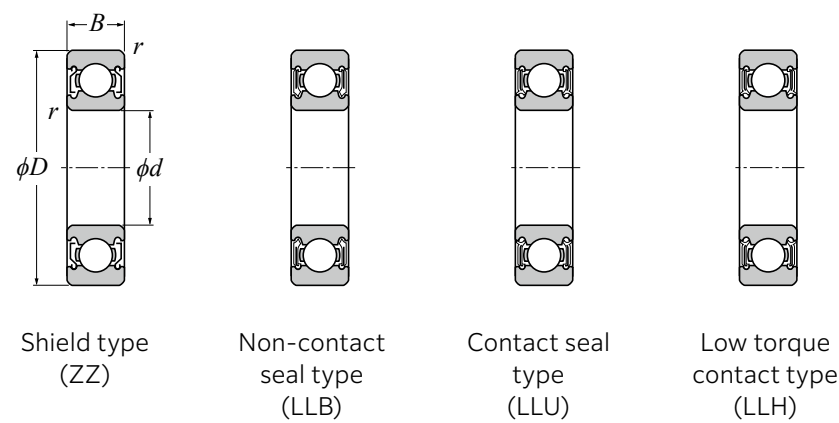
$\frac{f_0 \cdot F_a}{C_{0r}}$	e	$\frac{F_a}{F_r} \leq e$		$\frac{F_a}{F_r} > e$	
		X	Y	X	Y
0.172	0.19				2.30
0.345	0.22				1.99
0.689	0.26				1.71
1.03	0.28				1.55
1.38	0.30	1	0	0.56	1.45
2.07	0.34				1.31
3.45	0.38				1.15
5.17	0.42				1.04
6.89	0.44				1.00

Static equivalent radial load
 $P_{0r} = 0.6 F_r + 0.5 F_a$
When,
 $P_{0r} < F_r$ use $P_{0r} = F_r$.

d 22-35 mm

Boundary dimensions					Basic load rating		Fatigue load limit	Factor	Allowable speed				Bearing number					
					dynamic	static			min ⁻¹									
mm					kN		kN		Grease lubrication	Oil lubrication					Shielded or sealed type ²⁾ (See drawings)			
				r _{Ns}														
d	D	B	r _{s min} ¹⁾	Min.	C _r	C _{0r}	C _u	f ₀	ZZ,LLB	Z,LB	LLH	LLU						
									Z,LB		LH	LU						
22	44	12	0.6	0.5	10.4	5.05	0.395	13.9	17 000	20 000	13 000	10 000	60/22	ZZ	LLB	LLU	LLH	
	50	14	1	0.5	14.3	6.80	0.500	13.5	14 000	17 000	12 000	9 700	62/22	ZZ	LLB	LLU	LLH	
	56	16	1.1	0.5	20.4	9.25	0.725	12.4	13 000	15 000	11 000	9 200	63/22	ZZ	LLB	LLU	LLH	
25	37	7	0.3	0.3	4.75	2.95	0.208	16.1	18 000	21 000	—	10 000	6805JR	ZZ	LLB	LLU	—	
	42	9	0.3	0.3	7.80	4.55	0.345	15.4	16 000	19 000	11 700	9 800	6905	ZZ	LLB	LLU	LLH	
	47	12	0.6	0.5	11.2	5.85	0.380	14.5	15 000	18 000	11 000	9 400	6005	ZZ	LLB	LLU	LLH	
	52	15	1	0.5	15.5	7.85	0.550	13.9	13 000	15 000	11 000	8 900	6205	ZZ	LLB	LLU	LLH	
	62	17	1.1	0.5	23.5	10.9	0.855	12.6	12 000	14 000	9 700	8 100	6305	ZZ	LLB	LLU	LLH	
28	52	12	0.6	0.5	13.8	7.40	0.580	14.5	14 000	16 000	10 000	8 400	60/28	ZZ	LLB	LLU	LLH	
	58	16	1	0.5	19.8	9.75	0.720	13.4	12 000	14 000	9 700	8 100	62/28	ZZ	LLB	LLU	LLH	
	68	18	1.1	0.5	29.6	14.0	1.10	12.4	11 000	13 000	8 900	7 400	63/28	ZZ	LLB	LLU	LLH	
30	42	7	0.3	0.3	5.20	3.65	0.244	16.5	15 000	18 000	10 500	8 800	6806JR	ZZ	LLB	LLU	LLH	
	47	9	0.3	0.3	8.00	5.00	0.365	15.8	14 000	17 000	10 000	8 400	6906	ZZ	LLB	LLU	LLH	
	55	13	1	0.5	14.7	8.30	0.650	14.8	13 000	15 000	9 200	7 700	6006	ZZ	LLB	LLH	LLU	
	62	16	1	0.5	21.6	11.3	0.795	13.8	11 000	13 000	8 800	7 300	6206	ZZ	LLB	LLU	LLH	
	72	19	1.1	0.5	29.5	15.0	1.14	13.3	10 000	12 000	7 900	6 600	6306	ZZ	LLB	LLU	LLH	
32	58	13	1	0.5	13.1	8.05	0.615	15.4	12 000	15 000	8 700	7 200	60/32	ZZ	LLB	LLU	LLH	
	65	17	1	0.5	23.0	11.6	0.840	13.6	11 000	12 000	8 400	7 100	62/32	ZZ	LLB	LLU	LLH	
	75	20	1.1	0.5	33.0	16.9	1.30	13.1	9 500	11 000	7 700	6 500	63/32	ZZ	LLB	LLU	LLH	
35	47	7	0.3	0.3	5.45	4.05	0.268	16.4	13 000	16 000	—	7 600	6807JR	ZZ	LLB	LLU	—	
	55	10	0.6	0.5	10.6	6.85	0.495	15.8	12 000	15 000	8 500	7 100	6907	ZZ	LLB	LLU	LLH	
	62	14	1	0.5	17.7	10.3	0.805	14.8	12 000	14 000	8 200	6 800	6007	ZZ	LLB	LLU	LLH	
	72	17	1.1	0.5	28.4	15.3	1.09	13.8	9 800	11 000	7 600	6 300	6207	ZZ	LLB	LLU	LLH	
	80	21	1.5	0.5	37.0	19.1	1.47	13.1	8 800	10 000	7 300	6 000	6307	ZZ	LLB	LLU	LLH	

Dimension Table



Dynamic equivalent radial load
 $P_r = X F_r + Y F_a$

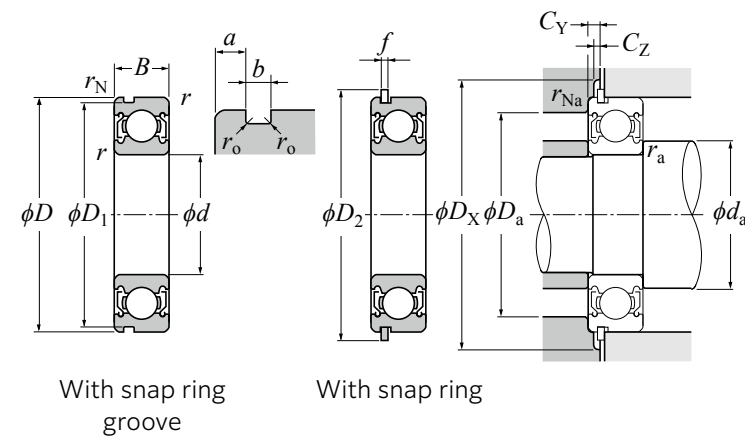
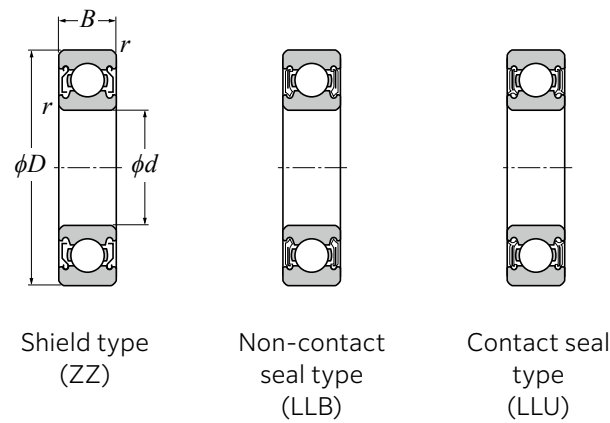
$\frac{f_0 \cdot F_a}{C_{0r}}$	e	$\frac{F_a}{F_r} \leq e$		$\frac{F_a}{F_r} > e$	
		X	Y	X	Y
0.172	0.19				2.30
0.345	0.22				1.99
0.689	0.26				1.71
1.03	0.28				1.55
1.38	0.30	1	0	0.56	1.45
2.07	0.34				1.31
3.45	0.38				1.15
5.17	0.42				1.04
6.89	0.44				1.00

Static equivalent radial load
 $P_{0r} = 0.6 F_r + 0.5 F_a$
When,
 $P_{0r} < F_r$ use $P_{0r} = F_r$.

d 40-60 mm

Boundary dimensions				Basic load rating		Fatigue	Factor	Allowable speed				Bearing number					
				dynamic	static	load limit		min ⁻¹									
mm				kN		kN		Grease	Oil								
								lubrication	lubrication								
				r _{Ns}				ZZ,LLB	Z,LB	LLH	LLU						
d	D	B	r _{s,min} ¹⁾	Min.	C _r	C _{0r}	C _u	f ₀	Z,LB	LH	LU	Shielded or sealed type ²⁾ (See drawings)					
40	52	7	0.3	0.3	5.65	4.40	0.291	16.3	12 000	14 000	8 000	6 700	6808JR	ZZ	LLB	LLU	LLH
	62	12	0.6	0.5	13.5	8.90	0.645	15.8	11 000	13 000	7 500	6 300	6908	ZZ	LLB	LLU	LLH
	68	15	1	0.5	18.6	11.5	0.890	15.2	10 000	12 000	7 300	6 100	6008	ZZ	LLB	LLU	LLH
	80	18	1.1	0.5	32.5	17.8	1.24	14.0	8 700	10 000	6 700	5 600	6208	ZZ	LLB	LLU	LLH
	90	23	1.5	0.5	45.0	24.0	1.83	13.2	7 800	9 200	6 400	5 300	6308	ZZ	LLB	LLU	LLH
45	58	7	0.3	0.3	5.95	4.95	0.325	16.1	11 000	12 000	—	5 900	6809JR	ZZ	LLB	LLU	—
	68	12	0.6	0.5	14.5	10.4	0.730	16.1	9 800	12 000	—	5 600	6909	ZZ	LLB	LLU	—
	75	16	1	0.5	23.2	15.1	1.16	15.3	9 200	11 000	6 500	5 400	6009	ZZ	LLB	LLU	LLH
	85	19	1.1	0.5	36.0	20.4	1.60	14.1	7 800	9 200	6 200	5 200	6209	ZZ	LLB	LLU	LLH
	100	25	1.5	0.5	58.5	32.0	2.50	13.1	7 000	8 200	5 600	4 700	6309	ZZ	LLB	LLU	LLH
50	65	7	0.3	0.3	7.30	6.10	0.405	16.1	9 600	11 000	—	5 300	6810JR	ZZ	LLB	LLU	—
	72	12	0.6	0.5	14.9	11.2	0.765	16.3	8 900	11 000	6 100	5 100	6910	ZZ	LLB	LLU	LLH
	80	16	1	0.5	24.2	16.6	1.24	15.5	8 400	9 800	6 000	5 000	6010	ZZ	LLB	LLU	LLH
	90	20	1.1	0.5	39.0	23.2	1.82	14.4	7 100	8 300	5 700	4 700	6210	ZZ	LLB	LLU	LLH
	110	27	2	0.5	68.5	38.5	2.99	13.2	6 400	7 500	5 000	4 200	6310	ZZ	LLB	LLU	LLH
55	72	9	0.3	0.3	9.75	8.10	0.540	16.2	8 700	10 000	—	4 800	6811JR	ZZ	LLB	LLU	—
	80	13	1	0.5	17.7	13.3	0.915	16.2	8 200	9 600	5 500	4 600	6911	ZZ	LLB	LLU	LLH
	90	18	1.1	0.5	31.5	21.2	1.62	15.3	7 700	9 000	—	4 500	6011	ZZ	LLB	LLU	—
	100	21	1.5	0.5	48.0	29.2	2.29	14.3	6 400	7 600	—	4 300	6211	ZZ	LLB	LLU	—
	120	29	2	0.5	79.5	45.0	3.50	13.2	5 800	6 800	—	3 900	6311	ZZ	LLB	LLU	—
60	78	10	0.3	0.3	12.7	10.6	0.705	16.3	8 000	9 400	—	4 400	6812	ZZ	LLB	LLU	—
	85	13	1	0.5	18.2	14.3	0.965	16.4	7 600	8 900	—	4 300	6912	ZZ	LLB	LLU	—
	95	18	1.1	0.5	32.5	23.2	1.73	15.6	7 000	8 300	—	4 100	6012	ZZ	LLB	LLU	—
	110	22	1.5	0.5	58.0	36.0	2.83	14.3	6 000	7 000	4 500	3 800	6212	ZZ	LLB	LLU	LLH
	130	31	2.1	0.5	90.5	52.0	4.10	13.2	5 400	6 300	—	3 600	6312	ZZ	LLB	LLU	—

Dimension Table



Dynamic equivalent radial load
 $P_r = X F_r + Y F_a$

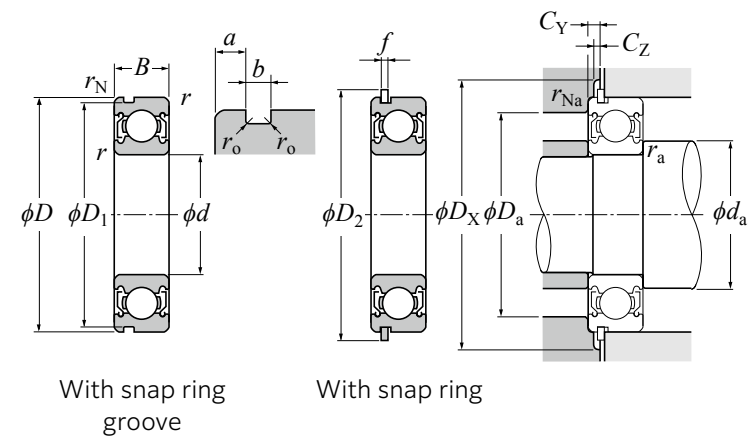
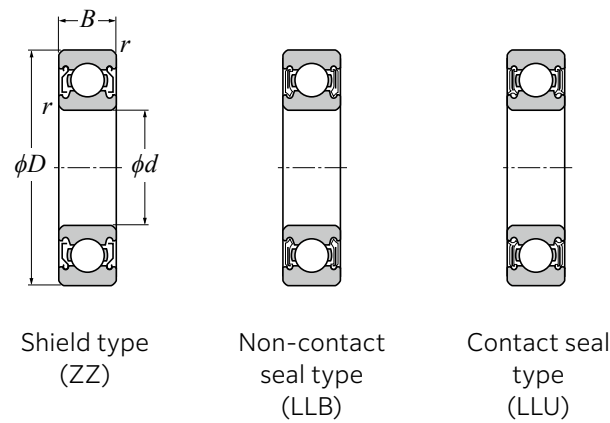
$\frac{f_0 \cdot F_a}{C_{0r}}$	e	$\frac{F_a}{F_r} \leq e$		$\frac{F_a}{F_r} > e$	
		X	Y	X	Y
0.172	0.19				2.30
0.345	0.22				1.99
0.689	0.26				1.71
1.03	0.28				1.55
1.38	0.30	1	0	0.56	1.45
2.07	0.34				1.31
3.45	0.38				1.15
5.17	0.42				1.04
6.89	0.44				1.00

Static equivalent radial load
 $P_{0r} = 0.6 F_r + 0.5 F_a$
When,
 $P_{0r} < F_r$ use $P_{0r} = F_r$.

d 65-85 mm

Boundary dimensions				Basic load rating		Fatigue	Factor	Allowable speed				Bearing number			
				dynamic	static	load limit		min ⁻¹							
mm				kN		kN		Grease lubrication	Oil lubrication						
			<i>r</i> _{Ns}					ZZ,LLB	Z,LB	LLU	Shielded or sealed type ²⁾ (See drawings)				
<i>d</i>	<i>D</i>	<i>B</i>	<i>r</i> _{s min} ¹⁾	Min.	<i>C</i> _r	<i>C</i> _{0r}	<i>C</i> _u	<i>f</i> ₀	Z,LB						
65	85	10	0.6	0.5	12.8	11.0	0.730	16.2	7 400	8 700	4 100	6813	ZZ	LLB	LLU
	90	13	1	0.5	19.3	16.1	1.07	16.6	7 000	8 200	4 000	6913	ZZ	LLB	LLU
	100	18	1.1	0.5	34.0	25.2	1.83	15.8	6 500	7 700	3 900	6013	ZZ	LLB	LLU
	120	23	1.5	0.5	63.5	40.0	3.15	14.4	5 500	6 500	3 600	6213	ZZ	LLB	LLU
	140	33	2.1	0.5	103	60.0	4.60	13.2	4 900	5 800	3 300	6313	ZZ	LLB	LLU
70	90	10	0.6	0.5	13.4	11.9	0.795	16.1	6 900	8 100	3 800	6814	ZZ	LLB	LLU
	100	16	1	0.5	26.3	21.2	1.45	16.3	6 500	7 700	3 700	6914	ZZ	LLB	LLU
	110	20	1.1	0.5	42.0	31.0	2.30	15.6	6 100	7 100	3 600	6014	ZZ	LLB	LLU
	125	24	1.5	0.5	69.0	44.0	3.45	14.5	5 100	6 000	3 400	6214	ZZ	LLB	LLU
	150	35	2.1	0.5	115	68.0	5.10	13.2	4 600	5 400	3 100	6314	ZZ	LLB	LLU
75	95	10	0.6	0.5	13.9	12.9	0.855	16.0	6 400	7 600	3 600	6815	ZZ	LLB	LLU
	105	16	1	0.5	27.0	22.6	1.52	16.5	6 100	7 200	3 500	6915	ZZ	LLB	LLU
	115	20	1.1	0.5	44.0	33.5	2.44	15.8	5 700	6 700	3 300	6015	ZZ	LLB	LLU
	130	25	1.5	0.5	73.5	49.5	3.80	14.7	4 800	5 600	3 200	6215	ZZ	LLB	LLU
	160	37	2.1	0.5	126	77.0	5.55	13.2	4 300	5 000	2 900	6315	ZZ	LLB	LLU
80	100	10	0.6	0.5	14.0	13.3	0.885	16.0	6 000	7 100	3 400	6816	ZZ	LLB	LLU
	110	16	1	0.5	27.6	24.0	1.59	16.6	5 700	6 700	3 200	6916	ZZ	LLB	LLU
	125	22	1.1	0.5	53.0	40.0	2.91	15.6	5 300	6 200	3 100	6016	ZZ	LLB	LLU
	140	26	2	0.5	80.5	53.0	3.95	14.6	4 500	5 300	3 000	6216	ZZ	LLB	LLU
	170	39	2.1	0.5	136	86.5	6.05	13.3	4 000	4 700	2 700	6316	ZZ	LLB	LLU
85	110	13	1	0.5	20.7	19.0	1.26	16.2	5 700	6 700	3 100	6817	ZZ	LLB	LLU
	120	18	1.1	0.5	35.5	29.6	1.99	16.4	5 400	6 300	3 000	6917	ZZ	LLB	LLU
	130	22	1.1	0.5	55.0	43.0	3.00	15.8	5 000	5 900	2 900	6017	ZZ	LLB	LLU
	150	28	2	0.5	92.0	64.0	4.60	14.7	4 200	5 000	2 800	6217	ZZ	LLB	LLU
	180	41	3	0.5	147	97.0	6.55	13.3	3 800	4 500	2 600	6317	ZZ	LLB	LLU

Dimension Table



Dynamic equivalent radial load
 $P_r = X F_r + Y F_a$

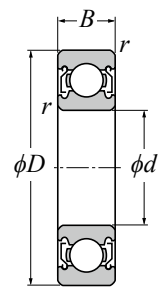
$\frac{f_0 \cdot F_a}{C_{0r}}$	e	$\frac{F_a}{F_r} \leq e$		$\frac{F_a}{F_r} > e$	
		X	Y	X	Y
0.172	0.19				2.30
0.345	0.22				1.99
0.689	0.26				1.71
1.03	0.28				1.55
1.38	0.30	1	0	0.56	1.45
2.07	0.34				1.31
3.45	0.38				1.15
5.17	0.42				1.04
6.89	0.44				1.00

Static equivalent radial load
 $P_{0r} = 0.6 F_r + 0.5 F_a$
When,
 $P_{0r} < F_r$ use $P_{0r} = F_r$.

d 90-120 mm

Boundary dimensions				Basic load rating		Fatigue	Factor	Allowable speed			Bearing number				
				dynamic	static	load limit		min ⁻¹							
mm				kN		kN		Grease lubrication	Oil lubrication						
			<i>r</i> _{s min¹}	<i>r</i> _{Ns}					ZZ,LLB	Z,LB	LLU	Shielded or sealed type ²⁾ (See drawings)			
<i>d</i>	<i>D</i>	<i>B</i>		Min.	<i>C</i> _r	<i>C</i> _{0r}	<i>C</i> _u	<i>f</i> ₀	Z,LB						
90	115	13	1	0.5	21.1	19.7	1.3	16.1	5 400	6 300	3 000	6818	ZZ	LLB	LLU
	125	18	1.1	0.5	36.5	31.5	2.05	16.5	5 100	6 000	2 900	6918	ZZ	LLB	LLU
	140	24	1.5	0.5	64.5	49.5	3.45	15.6	4 700	5 600	2 800	6018	ZZ	LLB	LLU
	160	30	2	0.5	106	71.5	5.00	14.5	4 000	4 700	2 600	6218	ZZ	LLB	LLU
	190	43	3	0.5	158	107	7.1	13.3	3 600	4 200	2 400	6318	ZZ	LLB	LLU
95	120	13	1	0.5	21.4	20.5	1.31	16.1	5 000	5 900	2 800	6819	ZZ	LLB	LLU
	130	18	1.1	0.5	37.5	33.5	2.10	16.6	4 800	5 700	2 800	6919	ZZ	LLB	LLU
	145	24	1.5	0.5	67.0	54.0	3.55	15.8	4 500	5 300	2 600	6019	ZZ	LLB	LLU
	170	32	2.1	0.5	121	82.0	5.55	14.4	3 700	4 400	2 500	6219	ZZ	LLB	LLU
	200	45	3	0.5	169	119	7.65	13.3	3 300	3 900	2 300	6319	ZZ	LLB	LLU
100	125	13	1	0.5	21.7	21.2	1.33	16.0	4 800	5 600	2 700	6820	ZZ	LLB	LLU
	140	20	1.1	0.5	45.5	39.5	2.44	16.4	4 500	5 300	2 600	6920	ZZ	LLB	LLU
	150	24	1.5	0.5	66.5	54.0	3.50	15.9	4 200	5 000	2 600	6020	ZZ	LLB	LLU
	180	34	2.1	0.5	135	93.0	6.15	14.4	3 500	4 200	2 300	6220	ZZ	LLB	LLU
	215	47	3	—	192	141	8.75	13.2	3 200	3 700	2 200	6320	ZZ	LLB	LLU
105	130	13	1	0.5	22.0	22.0	1.35	15.9	4 600	5 400	2 500	6821	ZZ	—	LLU
	145	20	1.1	0.5	47.0	42.0	2.52	16.5	4 300	5 100	2 500	6921	ZZ	LLB	LLU
	160	26	2	0.5	80.5	65.5	4.15	15.8	4 000	4 700	2 400	6021	ZZ	LLB	LLU
	190	36	2.1	0.5	147	105	6.75	14.4	3 400	4 000	2 300	6221	ZZ	LLB	LLU
	225	49	3	—	204	153	9.35	13.2	3 000	3 600	2 100	6321	ZZ	—	LLU
110	140	16	1	0.5	27.5	28.2	1.68	16.0	4 300	5 100	2 400	6822	ZZ	LLB	LLU
	150	20	1.1	0.5	48.5	44.5	2.60	16.6	4 100	4 800	2 400	6922	ZZ	LLB	LLU
	170	28	2	0.5	91.0	73.0	4.55	15.6	3 800	4 500	2 300	6022	ZZ	LLB	LLU
	200	38	2.1	0.5	160	117	7.35	14.3	3 200	3 800	2 200	6222	ZZ	LLB	LLU
	240	50	3	—	227	179	10.5	13.1	2 900	3 400	1 900	6322	ZZ	LLB	LLU
120	150	16	1	0.5	32.0	33.0	1.89	16.0	4 000	4 700	2 200	6824	ZZ	LLB	LLU
	165	22	1.1	0.5	59.0	54.0	3.05	16.5	3 800	4 400	2 100	6924	ZZ	—	LLU
	180	28	2	0.5	94.0	79.5	4.65	15.9	3 500	4 100	2 100	6024	ZZ	LLB	LLU

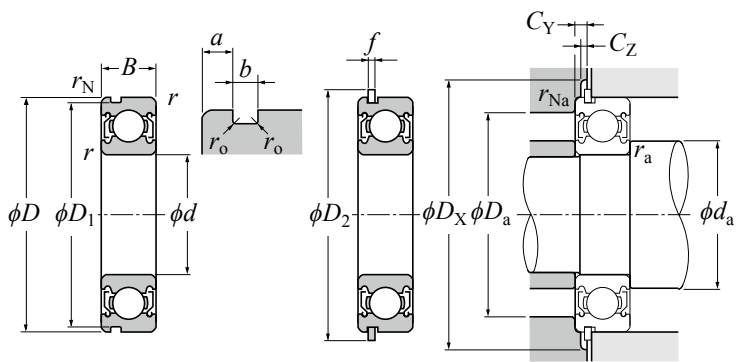
Dimension Table



Shield type
(ZZ)



Contact seal
type
(LLU)



With snap ring
groove

With snap ring

Dynamic equivalent radial load
 $P_r = X F_r + Y F_a$

$\frac{f_0 \cdot F_a}{C_{0r}}$	e	$\frac{F_a}{F_r} \leq e$		$\frac{F_a}{F_r} > e$	
		X	Y	X	Y
0.172	0.19				2.30
0.345	0.22				1.99
0.689	0.26				1.71
1.03	0.28				1.55
1.38	0.30	1	0	0.56	1.45
2.07	0.34				1.31
3.45	0.38				1.15
5.17	0.42				1.04
6.89	0.44				1.00

Static equivalent radial load

$P_{0r} = 0.6 F_r + 0.5 F_a$

When,
 $P_{0r} < F_r$ use $P_{0r} = F_r$.

d 120-170 mm

Boundary dimensions					Basic load rating		Fatigue load limit	Factor	Allowable speed			Bearing number		
mm					kN		kN		Grease lubrication	Oil lubrication				
d	D	B	$r_{s \min}^{1)}$	r_{Ns}	C_r	C_{0r}	C_u	f_0				Shielded or sealed type ²⁾ (See drawings)		
				Min.					ZZ	Z	LLU			
120	215	40	2.1	—	172	131	7.95	14.4	2 900	3 400	2 000	6224	ZZ	LLU
	260	55	3	—	229	185	10.5	13.5	2 600	3 100	1 700	6324	ZZ	LLU
130	165	18	1.1	0.5	41.0	41.0	2.25	16.1	3 700	4 300	2 000	6826	ZZ	LLU
	180	24	1.5	0.5	72.0	67.5	3.65	16.5	3 500	4 100	1 900	6926	ZZ	LLU
	200	33	2	0.5	118	101	5.70	15.8	3 200	3 800	1 900	6026	ZZ	LLU
	230	40	3	—	185	146	8.55	14.5	2 700	3 100	1 800	6226	ZZ	LLU
140	175	18	1.1	0.5	42.5	44.5	2.35	16.0	3 400	4 000	1 900	6828	ZZ	LLU
	190	24	1.5	0.5	74.0	71.5	3.70	16.6	3 200	3 800	1 800	6928	ZZ	LLU
	210	33	2	—	122	109	5.85	15.9	3 000	3 500	1 800	6028	ZZ	LLU
	250	42	3	—	184	150	8.40	14.8	2 500	2 900	1 600	6228	ZZ	LLU
150	190	20	1.1	0.5	53.0	55.0	2.80	16.1	3 100	3 700	1 700	6830	ZZ	LLU
	210	28	2	—	94.0	90.5	4.55	16.5	3 000	3 500	1 700	6930	ZZ	LLU
	225	35	2.1	—	139	126	6.55	15.9	2 800	3 200	1 700	6030	ZZ	LLU
	270	45	3	—	195	168	9.05	15.1	2 300	2 700	1 500	6230	ZZ	LLU
160	200	20	1.1	0.5	53.5	57.0	2.82	16.1	2 900	3 400	1 600	6832	ZZ	LLU
	220	28	2	—	96.5	96.0	4.65	16.6	2 800	3 300	1 600	6932	ZZ	LLU
	240	38	2.1	—	158	144	7.30	15.9	2 600	3 000	1 600	6032	ZZ	LLU
170	215	22	1.1	—	66.5	70.5	3.35	16.1	2 700	3 200	—	6834	ZZ	—
	230	28	2	—	95.0	95.5	4.50	16.5	2 600	3 100	—	6934	ZZ	—
	260	42	2.1	—	187	172	8.55	15.8	2 400	2 800	—	6034	ZZ	—

Bearing Number	Snap ring groove dimensions				Snap ring dimensions			Installation-related dimensions								Mass ³⁾
	mm				mm			mm								kg
Groove / Snap ring (See drawings)	D_1 Max.	a Max.	b Min.	r_o Max.	D_2 Max.	f Max.		d_a Min.	Max.	D_a Max.	D_X (approx.)	C_Y Max.	C_Z Min.	r_{as} Max.	r_{Nas} Max.	(approx.)
N NR	217.0	6.5	4.5	1	227.8	3.1		131	143	204	230	9.2	3.1	2	0.5	5.15
— —	—	—	—	—	—	—		133	162	247	—	—	—	2.5	—	12.4
N NR	161.8	3.3	1.9	0.6	171.5	1.7		136.5	139.5	158.5	173	4.7	1.7	1	0.5	0.8
N NR	176.8	3.7	1.9	0.6	186.5	1.7		138	144	172	188	5.1	1.7	1.5	0.5	1.52
N NR	193.65	5.69	3.5	0.6	212.9	3.1		139	148	191	215	8.4	3.1	2	0.5	3.16
N NR	222.0	6.5	4.5	1	242	3.5		143	158	217	244	9.6	3.5	2.5	0.5	5.82
N NR	171.8	3.3	1.9	0.6	181.5	1.7		146.5	150	168.5	183	4.7	1.7	1	0.5	0.85
N NR	186.8	3.7	1.9	0.6	196.5	1.7		148	154	182	198	5.1	1.7	1.5	0.5	1.62
— —	—	—	—	—	—	—		149	158	201	—	—	—	2	—	3.35
N NR	242.0	6.5	4.5	1	262	3.5		153	173	237	264	9.6	3.5	2.5	0.5	7.57
N NR	186.8	3.3	1.9	0.6	196.5	1.7		156.5	161	183.5	198	4.7	1.7	1	0.5	1.16
— —	—	—	—	—	—	—		159	167	201	—	—	—	2	—	2.47
— —	—	—	—	—	—	—		161	169	214	—	—	—	2	—	4.08
— —	—	—	—	—	—	—		163	188	257	—	—	—	2.5	—	9.41
N NR	196.8	3.3	1.9	0.6	206.5	1.7		166.5	171	193.5	208	4.7	1.7	1	0.5	1.23
— —	—	—	—	—	—	—		169	178	211	—	—	—	2	—	2.61
— —	—	—	—	—	—	—		171	183	229	—	—	—	2	—	5.05
— —	—	—	—	—	—	—		176.5	182	208.5	—	—	—	1	—	1.63
— —	—	—	—	—	—	—		179	188	221	—	—	—	2	—	2.74
— —	—	—	—	—	—	—		181	196	249	—	—	—	2	—	6.76

1) Smallest allowable dimension for chamfer dimension r .
2) This bearing number is for double sealed and double shielded type bearings, but single sealed and single shielded type are also available.

3) Does not include bearings with snap rings.

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NOTE : The appearance and specifications may be changed without prior notice if required to improve performance. Although care has been taken to assure the accuracy of the data compiled in this catalog, **NTN** does not assume any liability to any company or person for errors or omissions.